

Beyond Attenuation

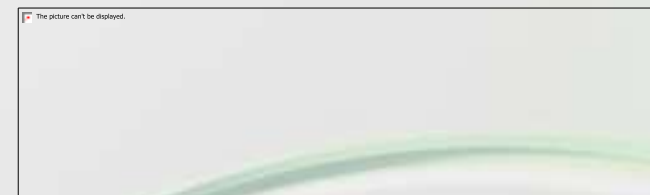
Asymptotic Rejection of Modelled Disturbances in Uncertain Linear Plants

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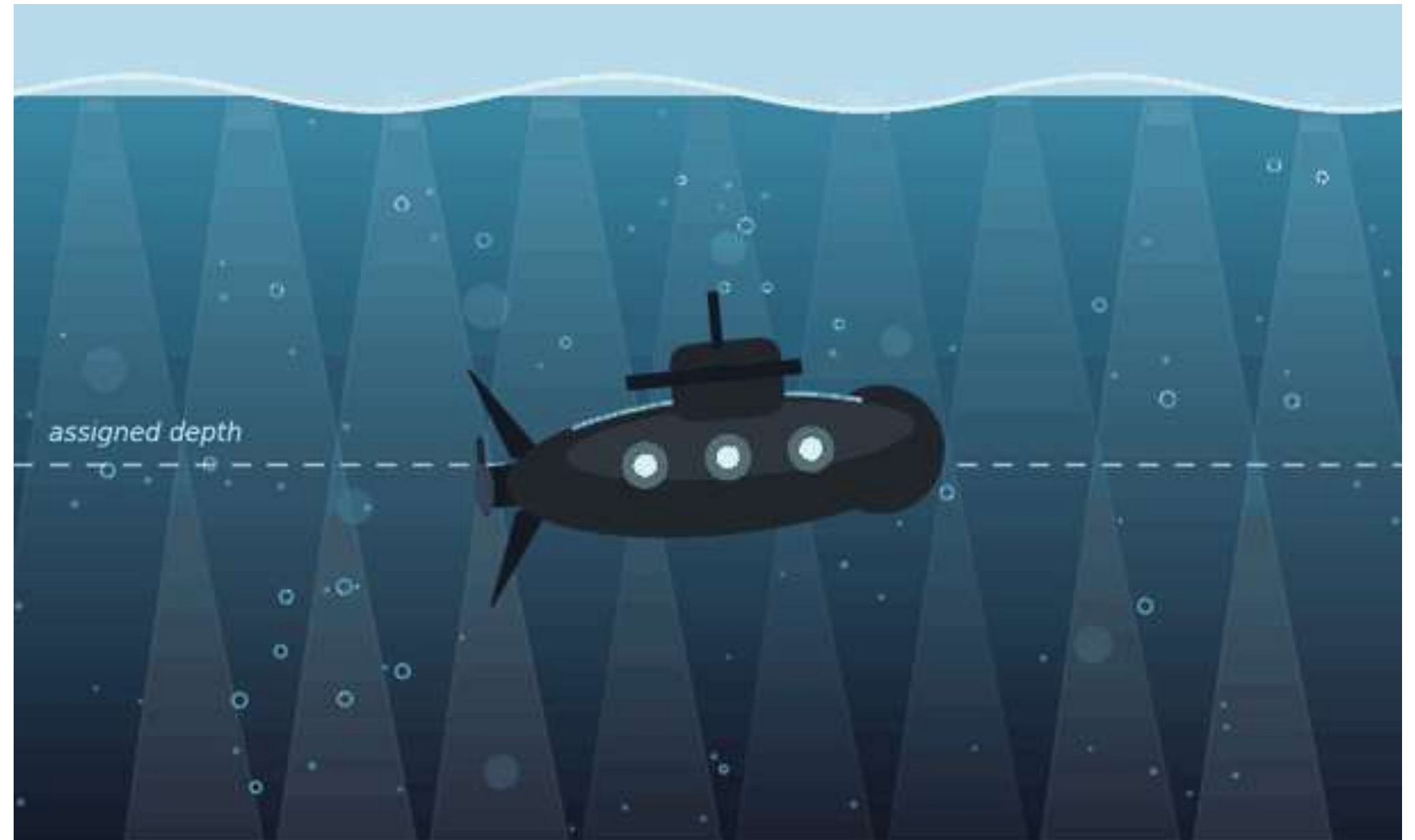


Why disturbance rejection?

A vehicle must hold a target depth.

A conventional controller only **attenuates** the disturbance so the vehicle never quite settles on the target.

This talk: how to **reject** modelled disturbances exactly when its model is not fully known.



Outline

① Existing works

- ✓ Conventional disturbance observers
- ✓ Internal-model-based disturbance observers

② Main result

- ✓ Problem formulation
- ✓ Internal-model-based design
- ✓ Stability analysis

③ Computer simulation

- ✓ Numerical example: mechanical positioning system

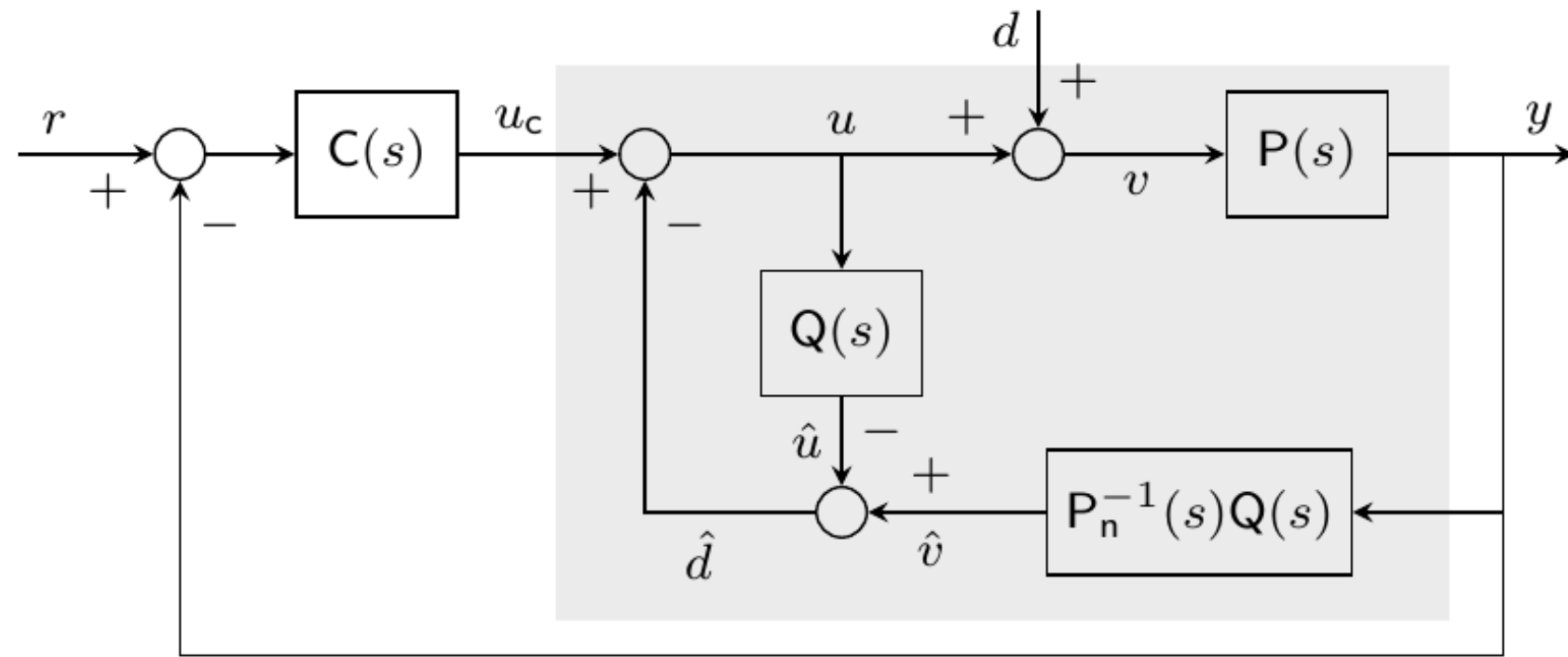
④ Real systems that use disturbance observers

- ✓ Hard-disk read/write head servo
- ✓ Linear actuator

⑤ Conclusion

Conventional disturbance observers (DOBs)

Existing Works



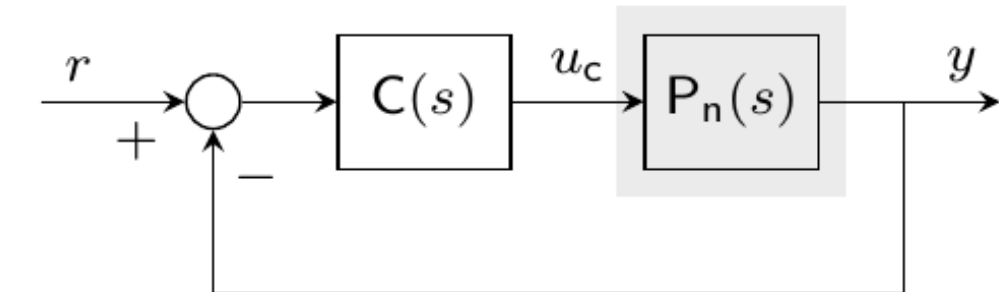
- P : Uncertain plant.
- P_n : Nominal plant.
- C : Outer-loop controller designed for P_n .
- Q : Low-pass filter with unity DC gain.
- d : external (**unmodeled and bounded**) disturbance.

• Closed-loop system:

$$y = \frac{PP_nC}{(1-Q)P_n + P(Q + P_nC)}r + \frac{(1-Q)PP_n}{(1-Q)P_n + P(Q + P_nC)}d.$$

• In low frequency range: $Q(j\omega) \approx 1$ and

$$y(j\omega) \approx \frac{P_n(j\omega)C(j\omega)}{1 + P_n(j\omega)C(j\omega)}r(j\omega).$$



Internal-model-based DOBs: Modeled disturbances

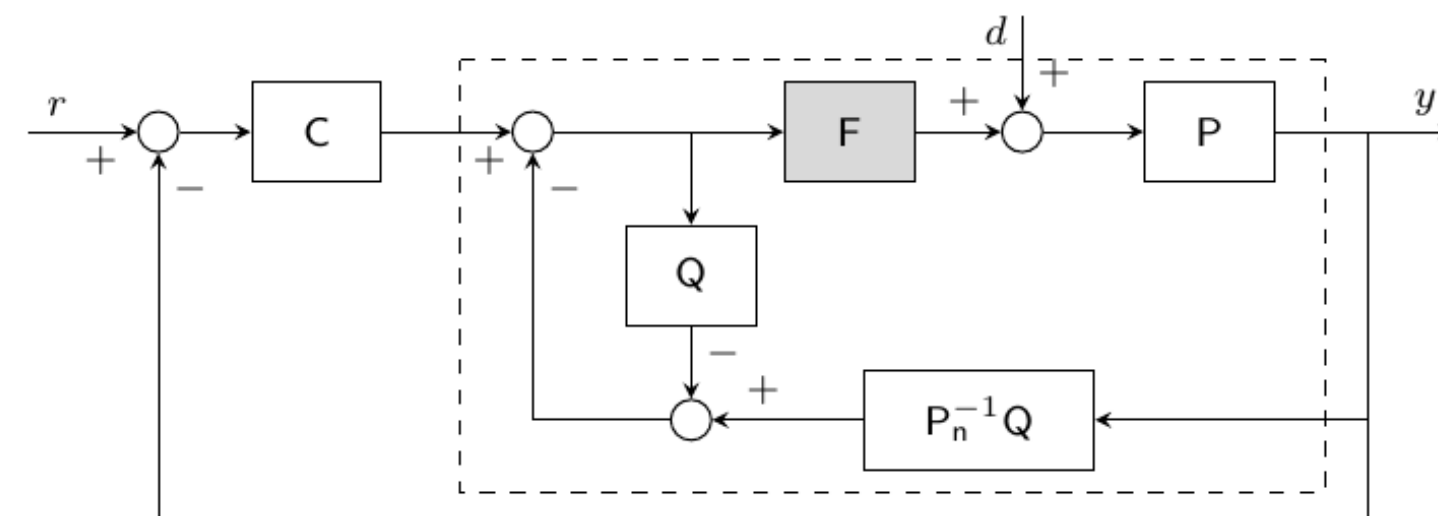
Existing Works

Adding an additional filter in feed-through path: [1]

- Disturbance under consideration:

$$\dot{w} = Sw, \quad d = [1 \ 0 \ \cdots \ 0]w.$$

- Increases the order and structural complexity of the DOB.

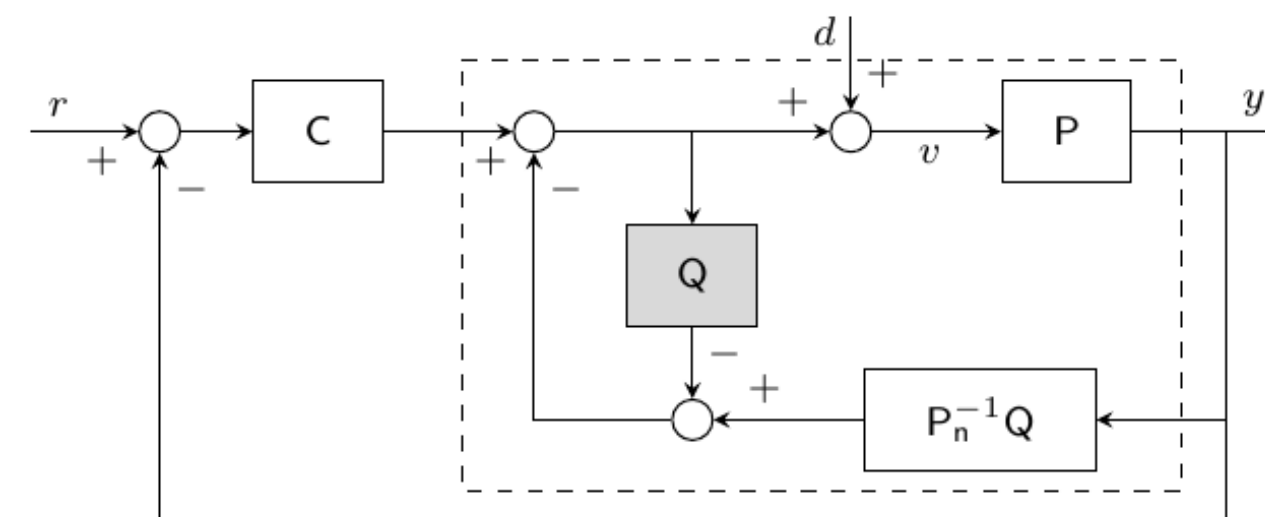


Embedding disturbance model in Q-filter: [2,3,4]

- Disturbance under consideration:

$$d(t) = \sum_{i=0}^{k_t} M_i t^i + \sum_{j=1}^{k_s} N_i \sin(\sigma_j t + \phi_j).$$

- Constraint: $\sigma_i \neq \sigma_j$.
Otherwise, the DOB design is infeasible.
- Coefficients of the numerator of $Q(s)$ -filter depend on τ *implicitly*.



[1] Kim, Park, & Shim, "Robust disturbance rejection for uncertain linear system with TV exosystem: A DOB approach," *AUT*, 2025.

[2] Joo, Park, Back, & Shim, "Embedding internal model in DOB with robust stability," *TAC*, 2016.

[3] Park, Shim, & Joo, "Recovering nominal tracking performance in an asymptotic sense for uncertain linear systems," *SIAM*, 2018.

[4] Kim, Joo, & Park, "Design of HGO-based High-order internal model DOB," *Proc. of ICCAS*, 2024.

Problem formulation: Plant and modeled disturbance

Main Result

Plant P in Byrnes-Isidori's normal form:

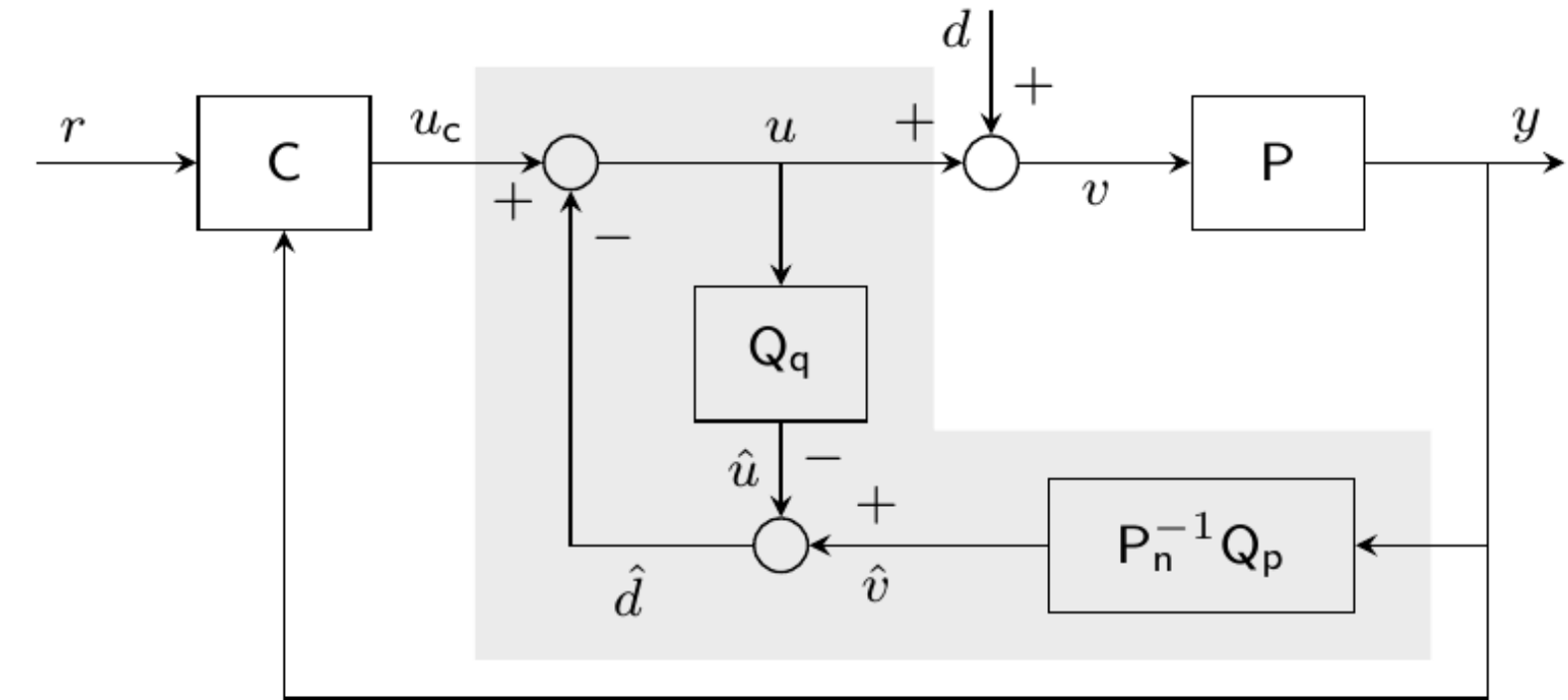
$$\dot{x} = A_\nu x + B_\nu \left(\phi^\top x + g(u + d) \right),$$

$$y = C_\nu x,$$

where $x \in \mathbb{R}^\nu$ and $u, y, d \in \mathbb{R}$.

Assumption (Plant)

- Uncertain $\phi \in \mathbb{R}^\nu$ lies in a known compact set.
- $g \in [g_{\min}, g_{\max}]$, where $g_{\min} > 0$ and $g_{\max}$ are known.



$$A_\nu := \begin{bmatrix} 0_{\nu-1} & I_{\nu-1} \\ 0 & 0_{\nu-1}^\top \end{bmatrix}, \quad B_\nu := \begin{bmatrix} 0_{\nu-1} \\ 1 \end{bmatrix}, \quad C_\nu := \begin{bmatrix} 1 & 0_{\nu-1}^\top \end{bmatrix}.$$

Modeled disturbance:

$$\begin{aligned} \dot{w} &= Sw, \\ d &= Rw, \end{aligned} \quad w \in \mathbb{R}^m.$$

Assumption (Disturbance)

- S and R are unknown but $\operatorname{Re} \lambda_i(S) \geq 0$ for $i = 1, \dots, m$.
- The minimal polynomial of S is known, where

$$\Delta_w(s) := s^\mu + a_{\mu-1}s^{\mu-1} + \dots + a_1s + a_0.$$

Problem formulation: Nominal systems and the problem

Main Result

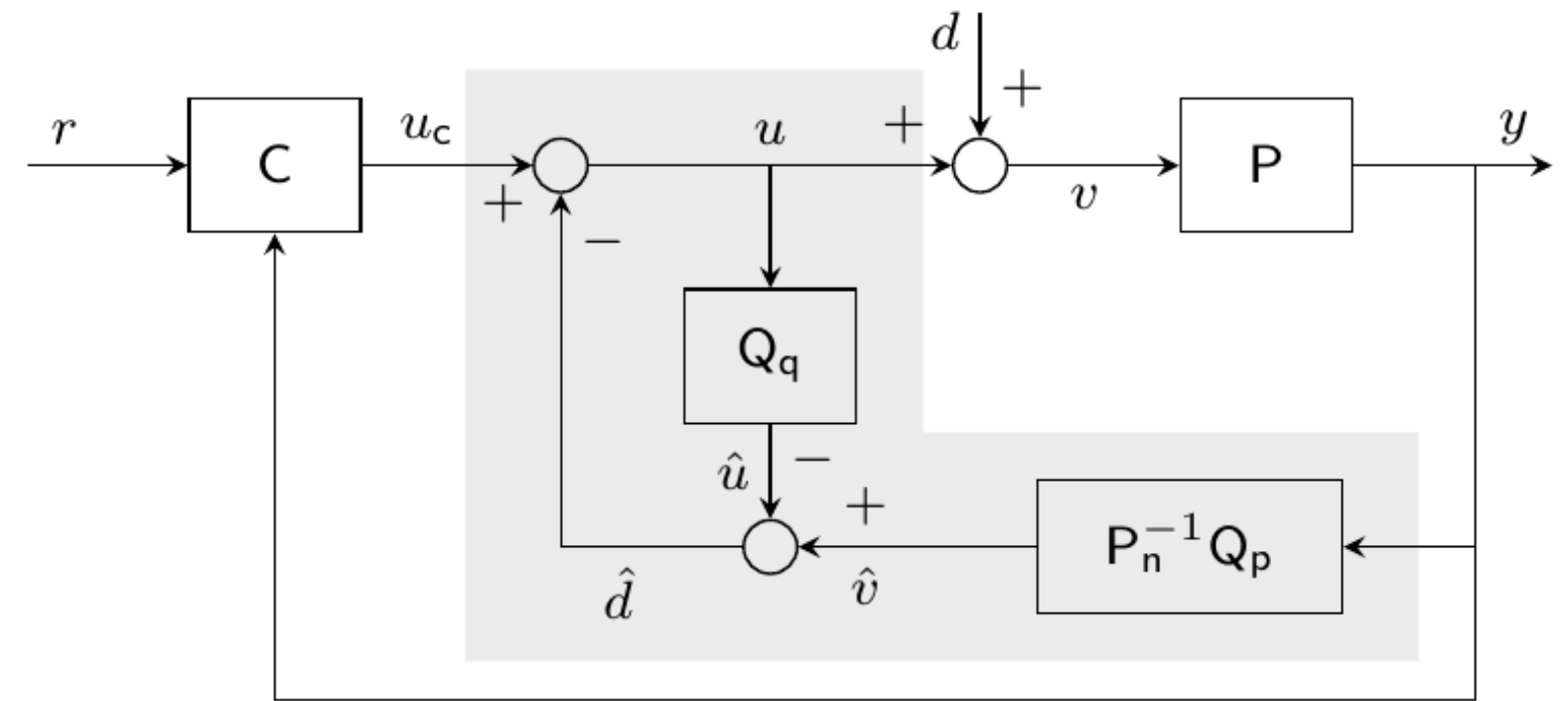
Nominal counterpart P_n of P :

$$\dot{x}_n = A_\nu x_n + B_\nu \left(\phi_n^\top x_n + g_n u_n \right),$$

$$y_n = C_\nu x_n.$$

Note: Nominal plant contains no uncertainty and disturbance.

Nominal controller: $\dot{\zeta}_n = F\zeta_n + G_1 y_n + G_2 r,$
 $u_n = J\zeta_n + K_1 y_n + K_2 r.$



Assumption (Nominal closed-loop system)

The nominal closed-loop system with $\chi_n := [x_n; \zeta_n]$ is asymptotically stable under $r \equiv 0$, i.e., $\mathcal{A}_s$ is Hurwitz.

$$\dot{\chi}_n = \mathcal{A}_s \chi_n + \mathcal{B}_s r := \begin{bmatrix} A_\nu + B_\nu (\phi_n^\top + g_n K_1 C_\nu) & g_n B_\nu J \\ G_1 C_\nu & F \end{bmatrix} \chi_n + \begin{bmatrix} g_n B_\nu K_2 \\ G_2 \end{bmatrix} r.$$

DOB problem

Design a DOB (Q_q and $P_n^{-1} Q_p$) s.t. the actual closed-loop system is asymptotically stable under $r \equiv 0$.

Internal-model-based design: $P_n^{-1}Q_p$

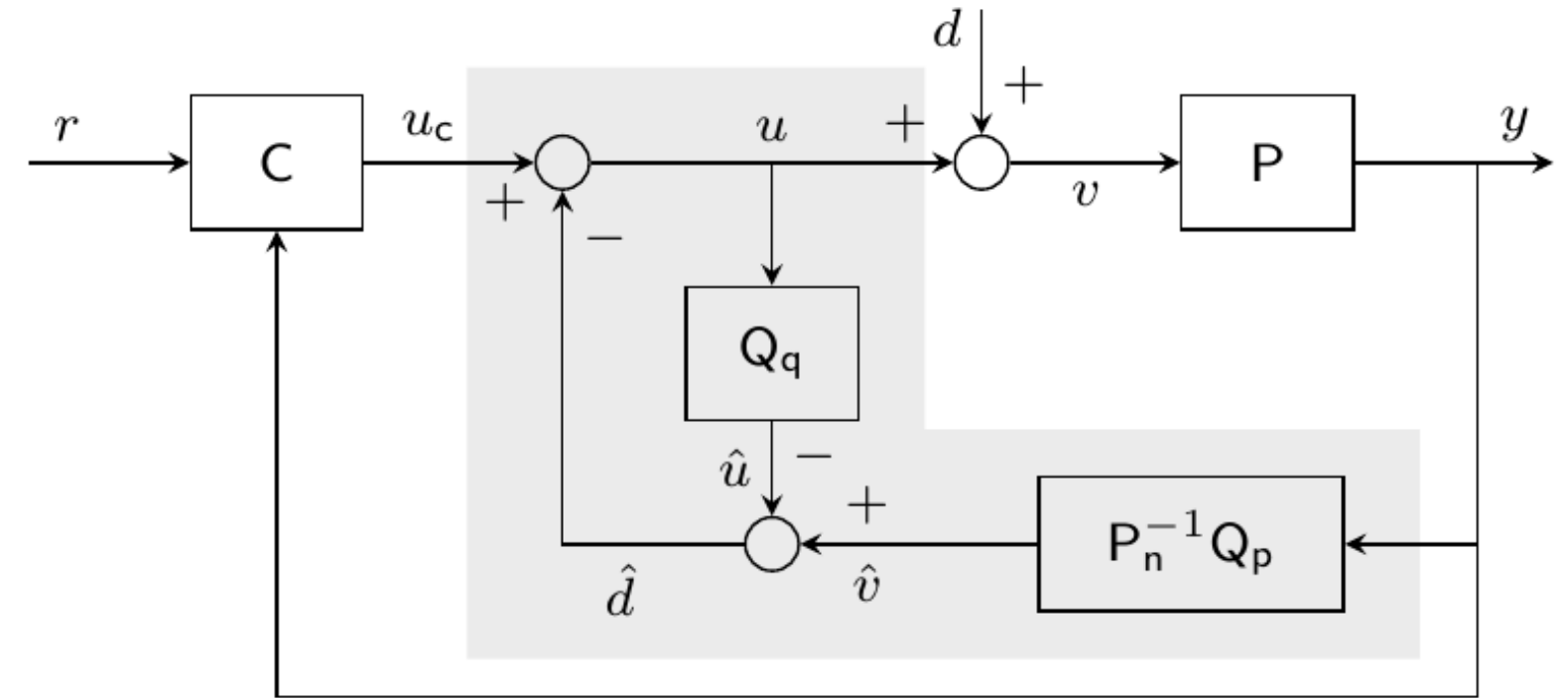
Main Result

Nominal plant P_n :

$$\begin{aligned} \dot{x}_n &= A_\nu x_n + B_\nu \left(\phi_n^\top x_n + g_n u_n \right) \Rightarrow \begin{aligned} y_n &= x_{n,1} \\ \dot{x}_{n,i} &= x_{n,i+1} \\ \dot{x}_{n,\nu} &= \phi_n^\top x_n + g_n u_n \end{aligned} \\ y_n &= C_\nu x_n \end{aligned}$$

Nominal inverse P_n^{-1} :

$$u_n = \frac{1}{g_n} \left(y_n^{(\nu)} - \phi_n^\top [y_n; \dot{y}_n; \cdots; y_n^{(\nu-1)}] \right).$$



Implementation of $P_n^{-1}Q_p$:

$$\begin{aligned} \dot{p} &= A_\nu p + H(y - C_\nu p), \\ \hat{v} &= \frac{1}{g_n} \left(\dot{p}_\nu - \phi_n^\top p \right), \end{aligned} \quad H := \left[\frac{\beta_{\nu-1}}{\tau}; \frac{\beta_{\nu-2}}{\tau^2}; \cdots; \frac{\beta_0}{\tau^\nu} \right], \quad A_\nu = \begin{bmatrix} 0_{\nu-1} & I_{\nu-1} \\ 0 & 0_{\nu-1}^\top \end{bmatrix}, \quad C_\nu = [1 \quad 0_{\nu-1}^\top],$$

where $p := [p_1; \cdots; p_\nu] \in \mathbb{R}^\nu$, $\hat{v} \in \mathbb{R}$, and $\Delta_p(s) = s^\nu + \beta_{\nu-1}s^{\nu-1} + \cdots + \beta_1s + \beta_0$ is a Hurwitz polynomial.

Internal-model-based design: Q_q and C

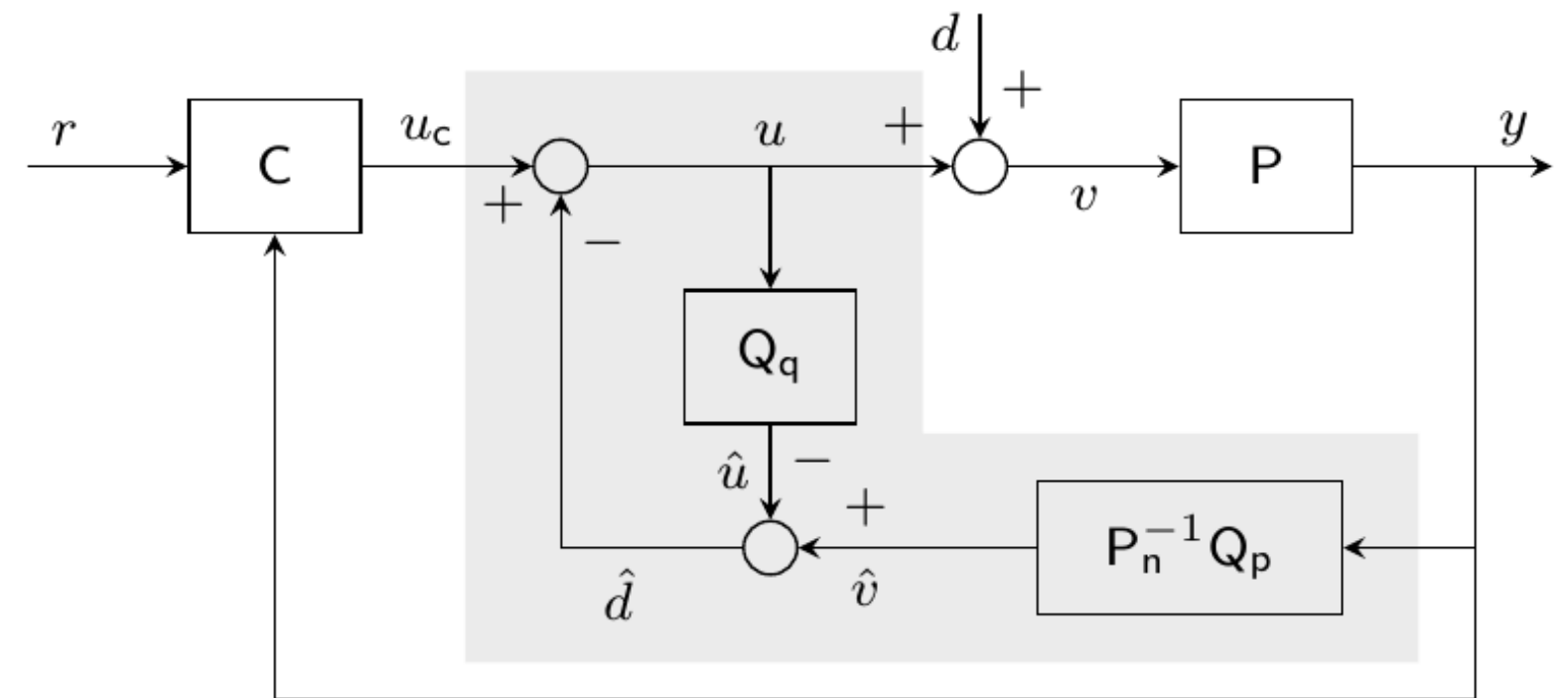
Main Result

Nominal controller: $\dot{\zeta}_n = F\zeta_n + G_1 y_n + G_2 r$,
 $u_n = J\zeta_n + K_1 y_n + K_2 r$.

Outer-loop controller C :

$$\dot{\zeta} = F\zeta + G_1 y + G_2 r,$$

$$u_c = J\zeta + K_1 y + K_2 r.$$



Design of Q_q : With $q := [q_1; \dots; q_\mu] \in \mathbb{R}^\mu$,

$$\begin{aligned} \dot{q} &= A_q q + B_q u, \\ \hat{u} &= C_\mu q, \\ u &= u_c - (\hat{v} - \hat{u}), \end{aligned} \quad A_q := \begin{bmatrix} -\frac{\alpha_{\mu-1}}{\tau^{\mu-1}} & 1 & 0 & \dots & 0 \\ -\frac{\alpha_{\mu-2}}{\tau^{\mu-2}} & 0 & 1 & \dots & 0 \\ \vdots & \vdots & \vdots & \ddots & \vdots \\ -\frac{\alpha_1}{\tau^{\mu-1}} & 0 & 0 & \dots & 1 \\ -\frac{\alpha_0}{\tau^\mu} & 0 & 0 & \dots & 0 \end{bmatrix}, \quad B_q := \begin{bmatrix} \frac{\alpha_{\mu-1}}{\tau^{\mu-1}} & -a_{\mu-1} \\ \frac{\alpha_{\mu-2}}{\tau^{\mu-2}} & -a_{\mu-2} \\ \vdots & \vdots \\ \frac{\alpha_1}{\tau^{\mu-1}} & -a_1 \\ \frac{\alpha_0}{\tau^\mu} & -a_0 \end{bmatrix}, \quad C_\mu := [1 \quad 0_{\mu-1}^\top],$$

where $\Delta_q(s) = s^\mu + \alpha_{\mu-1}s^{\mu-1} + \dots + \alpha_1 s + \alpha_0$ is a Hurwitz polynomial and
 $\Delta_w(s) = s^\mu + a_{\mu-1}s^{\mu-1} + \dots + a_1 s + a_0$ is the minimal polynomial of S .

Stability analysis: Internal model principle

Main Result

Recall that $\dot{w} = Sw$, $d = Rw$, and $\Delta_w(s) = s^\mu + a_{\mu-1}s^{\mu-1} + \dots + a_1s + a_0$.

Define $\Pi_1 := -R$, $\Pi_i := \Pi_{i-1}S + a_{\mu+1-i}\Pi_1$ for $i = 2, \dots, \mu$, and $\Pi := [\Pi_1; \dots; \Pi_\mu] \in \mathbb{R}^{\mu \times m}$.

Then, since $\Delta_w(S) = 0$ and $\Pi_{i-1}S = -a_{\mu+1-i}\Pi_1 + \Pi_i$,

$$\Pi_1 S = -a_{\mu-1}\Pi_1 + \Pi_2,$$

$$\vdots$$

$$\Pi_{\mu-1}S = -a_1\Pi_1 + \Pi_\mu,$$

$$\Pi_\mu S + a_0\Pi_1 = \Pi_{\mu-1}S^2 + a_1\Pi_1 S + a_0\Pi_1 = \dots$$

$$= \Pi_1 (S^\mu + a_{\mu-1}S^{\mu-1} + \dots + a_1S + a_0I) = 0,$$

$$\Pi_1 S = -a_{\mu-1}\Pi_1 + \Pi_2,$$

$$\vdots$$

$$\Pi_{\mu-1}S = -a_1\Pi_1 + \Pi_\mu,$$

$$\Pi_\mu S = -a_0\Pi_1.$$

Matrix identity (internal model principle):

$$\begin{bmatrix} \Pi_1 \\ \vdots \\ \Pi_{\mu-1} \\ \Pi_\mu \end{bmatrix} S = \begin{bmatrix} -a_{\mu-1} & 1 & 0 & \dots & 0 \\ \vdots & \vdots & \vdots & \ddots & \vdots \\ -a_1 & 0 & 0 & \dots & 1 \\ -a_0 & 0 & 0 & \dots & 0 \end{bmatrix} \begin{bmatrix} \Pi_1 \\ \vdots \\ \Pi_{\mu-1} \\ \Pi_\mu \end{bmatrix} \iff \begin{aligned} \Pi S &= (A_\mu - aC_\mu) \Pi, \\ -R &= C_\mu \Pi \quad (= [1 \quad 0 \quad \dots \quad 0] \Pi), \end{aligned}$$

where $a := [a_{\mu-1}; \dots; a_1; a_0] \in \mathbb{R}^\mu$ and $A_\mu := [0_{\mu-1} \quad I_{\mu-1}; 0 \quad 0_{\mu-1}^\top]$.

Stability analysis: Standard form of singular perturbations

Main Result

Closed-loop system:

$$\begin{aligned} \dot{x} &= A_\nu x + B_\nu(\phi^\top x + g(u + d)), & y &= C_\nu x, \\ \dot{\zeta} &= F\zeta + G_1 y + G_2 r, & u_c &= J\zeta + K_1 y + K_2 r, \\ \dot{p} &= A_\nu p + H(y - C_\nu p), & \hat{v} &= \frac{1}{g_n} (\dot{p}_\nu - \phi_n^\top p), \\ \dot{q} &= A_q q + B_q u, & u &= u_c + \hat{u} - \hat{v}, \quad \hat{u} = C_\mu q. \end{aligned}$$

Coordinate changes: $\hat{q} := \frac{1}{\tau} \mathcal{T}(q - \Pi w)$ and $\hat{p} := \Upsilon^{-1}(x - p)$,
 where $\mathcal{T} := \text{diag}\{\tau, \tau^2, \dots, \tau^\mu\}$ and $\Upsilon := \text{diag}\{\tau^\nu, \tau^{\nu-1}, \dots, \tau\}$.

Standard form:

$$\begin{aligned} \dot{x} &= \left(A_\nu + B_\nu \phi^\top \right) x - \frac{g}{g_n} \beta_0 B_\nu C_\nu \hat{p} + g B_\nu C_\mu \hat{q} + g B_\nu f(x, \zeta, r, \hat{p}, \tau), \\ \dot{\zeta} &= F\zeta + G_1 C_\nu x + G_2 r, \\ \tau \begin{bmatrix} \dot{\hat{p}} \\ \dot{\hat{q}} \end{bmatrix} &= \mathcal{A}_f \begin{bmatrix} \hat{p} \\ \hat{q} \end{bmatrix} + \begin{bmatrix} B_\nu \phi^\top x + g B_\nu f(x, \zeta, r, \hat{p}, \tau) \\ \alpha f(x, \zeta, r, \hat{p}, \tau) - \tau a \left(C_\mu \hat{q} - \frac{\beta_0}{g_n} C_\nu \hat{p} + f(x, \zeta, r, \hat{p}, \tau) \right) \end{bmatrix}, \end{aligned}$$

where $f(\cdot) := J\zeta + K_1 C_\nu x + K_2 r + \frac{1}{g_n} \phi_n^\top (x - \Upsilon \hat{p})$, $\alpha := [\alpha_{\mu-1}; \dots; \alpha_1; \alpha_0]$, $\beta := [\beta_{\nu-1}; \dots; \beta_1; \beta_0]$, and

$$\mathcal{A}_f := \begin{bmatrix} A_\nu - \beta C_\nu - \frac{g}{g_n} \beta_0 B_\nu C_\nu & g B_\nu C_\mu \\ -\frac{\beta_0}{g_n} \alpha C_\nu & A_\mu \end{bmatrix}.$$

Note: No disturbance signal w appears in the above equation.

Stability analysis: Quasi-steady-states

Main Result

Standard form:

$$\begin{aligned}\dot{x} &= \left(A_\nu + B_\nu \phi^\top \right) x - \frac{g}{g_n} \beta_0 B_\nu C_\nu \hat{p} + g B_\nu C_\mu \hat{q} + g B_\nu f(x, \zeta, r, \hat{p}, \tau), \\ \dot{\zeta} &= F\zeta + G_1 C_\nu x + G_2 r, \\ \tau \begin{bmatrix} \dot{\hat{p}} \\ \dot{\hat{q}} \end{bmatrix} &= \mathcal{A}_f \begin{bmatrix} \hat{p} \\ \hat{q} \end{bmatrix} + \begin{bmatrix} B_\nu \phi^\top x + g B_\nu f(x, \zeta, r, \hat{p}, \tau) \\ \alpha f(x, \zeta, r, \hat{p}, \tau) - \tau a \left(C_\mu \hat{q} - \frac{\beta_0}{g_n} C_\nu \hat{p} + f(x, \zeta, r, \hat{p}, \tau) \right) \end{bmatrix}.\end{aligned}$$

Quasi-steady-states: By letting $\tau = 0$,

$$\begin{aligned}\hat{p}_i^* &= \frac{\beta_{\nu-i+1}}{\beta_0} g_n \left(J\zeta + K_1 C_\nu x + K_2 r + \frac{1}{g_n} \phi_n^\top x \right), & i &= 1, \dots, \nu, \quad \beta_\nu := 1, \\ \hat{q}_1^* &= \frac{1}{g} (\phi_n - \phi)^\top x + \frac{g_n}{g} (J\zeta + K_1 C_\nu x + K_2 r), \\ \hat{q}_j^* &= 0, & j &= 2, \dots, \mu.\end{aligned}$$

Note: Quasi-steady-states do not depend on the signal w !

Stability analysis: Reduced and boundary layer systems

Main Result

Reduced system: With $\chi := [x; \zeta]$,

$$\dot{\chi} = \begin{bmatrix} A_\nu + B_\nu (\phi_n^\top + g_n K_1 C_\nu) & g_n B_\nu J \\ G_1 C_\nu & F \end{bmatrix} \chi + \begin{bmatrix} g_n B_\nu K_2 \\ G_2 \end{bmatrix} r = \mathcal{A}_s \chi + \mathcal{B}_s r.$$

Nominal closed-loop system: $\dot{\chi}_n = \mathcal{A}_s \chi_n + \mathcal{B}_s r$.

Boundary layer system: By letting $\hat{t} := \frac{t}{\tau}$ and $\tau = 0$,

$$\begin{bmatrix} \hat{p}' \\ \hat{q}' \end{bmatrix} = \begin{bmatrix} A_\nu - \beta C_\nu - \frac{g}{g_n} \beta_0 B_\nu C_\nu & g B_\nu C_\mu \\ -\frac{\beta_0}{g_n} \alpha C_\nu & A_\mu \end{bmatrix} \begin{bmatrix} \hat{p} \\ \hat{q} \end{bmatrix} = \mathcal{A}_f \begin{bmatrix} \hat{p} \\ \hat{q} \end{bmatrix}.$$

Lemma (Stability of boundary layer system)

The characteristic polynomial of $\mathcal{A}_f$ is given by

$$\begin{aligned} \Delta_f(s) &= s^{\nu+\mu} + \beta_{\nu-1} s^{\nu+\mu-1} + \dots + \beta_1 s^{\mu+1} + \beta_0 s^\mu + \frac{g}{g_n} \beta_0 (s^\mu + \alpha_{\mu-1} s^{\mu-1} + \dots + \alpha_1 s + \alpha_0) \\ &= s^\mu \Delta_p(s) + \frac{g}{g_n} \beta_0 \Delta_q(s). \end{aligned}$$

- $\Delta_f(s)$ can always be made Hurwitz for all $g \in [g_{\min}, g_{\max}]$ by suitably choosing α and β .

Stability analysis: Main theorem

Main Result

Theorem (Main theorem)

Suppose that

- *the nominal closed-loop system is asymptotically stable (stability of reduced system);*
- *$\Delta_p(s)$ and $\Delta_q(s)$ are Hurwitz polynomial; and*
- *$\Delta_f(s)$ is a Hurwitz polynomial (stability of boundary layer system).*

Then under $r \equiv 0$, $\exists \tau^ > 0$ such that for all $\tau \in (0, \tau^*)$,*

- *the actual closed-loop system is asymptotically stable; and*
- *the following hold.*

$$\lim_{t \rightarrow \infty} \left\| \begin{bmatrix} x(t) \\ p(t) \end{bmatrix} \right\| = 0, \quad \lim_{t \rightarrow \infty} \|q(t) - \Pi w(t)\| = 0.$$

- Embedding of an internal model of disturbance into the Q_q -filter is confirmed by the asymptotic behavior.
- Since $u + d = u_c + \hat{u} - \hat{v} + d$, $\hat{u} = C_\mu q$, $d = R w$, and $-R = C_\mu \Pi$, we have

$$u(t) + d(t) = u_c(t) + C_\mu q(t) - \hat{v}(t) + d(t) \longrightarrow u_c(t) + C_\mu \Pi w(t) - \hat{v}(t) + d(t) = u_c(t) - \hat{v}(t).$$

- It is not necessary for the Q_q -dynamics to contain a zero eigenvalue.

Mechanical positioning system

Computer Simulation

Plant P [5,6]:

$$\dot{x} = \begin{bmatrix} 0 & 1 \\ 0 & -\mathfrak{B}/\mathfrak{J} \end{bmatrix} x + \begin{bmatrix} 0 \\ 1/\mathfrak{J} \end{bmatrix} (u + d), \quad y = \begin{bmatrix} 1 & 0 \end{bmatrix} x,$$

where $\mathfrak{J} \in [0.2, 0.4]$ V/(m/s²) and $\mathfrak{B} \in [0.1, 0.2]$ V/(m/s).

Disturbance model:

$$\dot{w} = Sw, \quad d = Rw, \quad S := \begin{bmatrix} S_\sigma & I_2 \\ 0 & S_\sigma \end{bmatrix}, \quad S_\sigma := \begin{bmatrix} 0 & \sigma \\ -\sigma & 0 \end{bmatrix},$$

where $R \in [-1, 1]^2$ and $\sigma = 30$ rad/s.

Nominal plant P_n : $\mathfrak{J}_n = 0.3$ V/(m/s²) and $\mathfrak{B}_n = 0.15$ V/(m/s).

Nominal controller: With $r(t) = 0.03$ m for $t \geq 0$,

$$\dot{\zeta}_n = -100.5\zeta_n - 5525y_n + 500r, \quad u_n = -15\zeta_n - 900y_n + 150r.$$

Q_p -filter: $\Delta_p(s) = s^2 + 2s + 1$.

Q_q -filter: $\Delta_q(s) = s^4 + 2s^3 + 0.2s^2 + 0.04s + 0.0005$.

$\implies \Delta_f(s)$ is Hurwitz for all $g \in [2.5, 5]$.

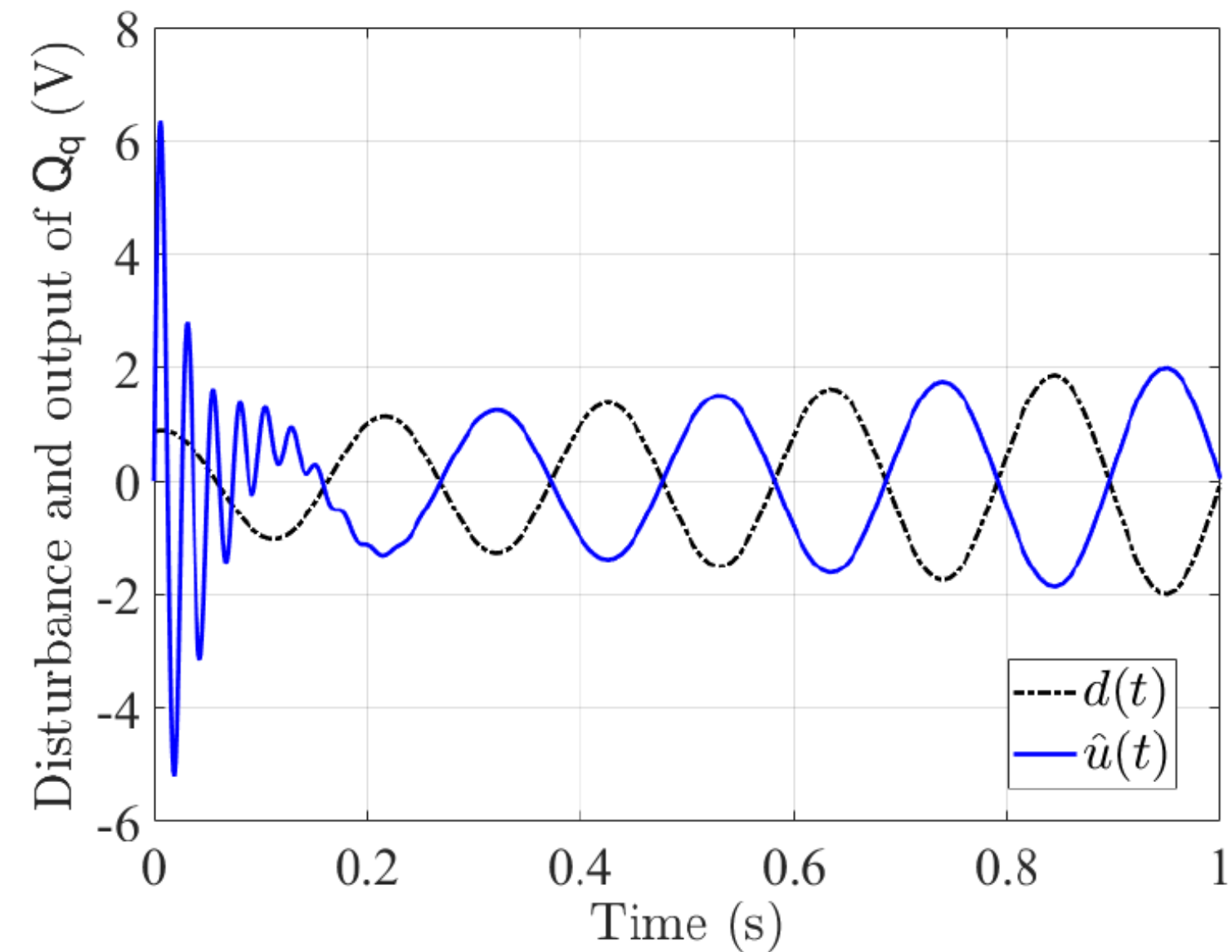
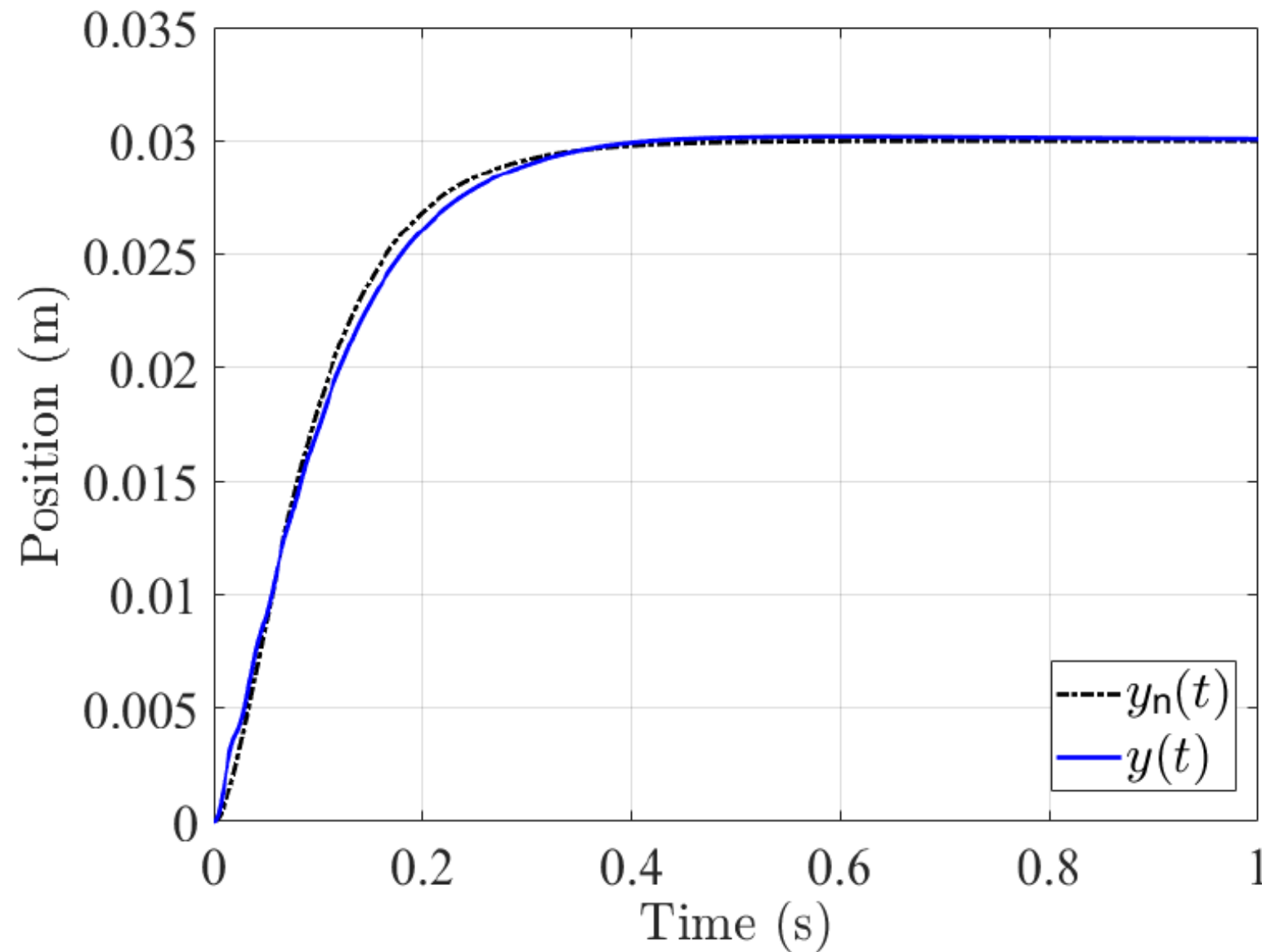
[5] Kim & Chung, "Advanced disturbance observer design for mechanical positioning systems," *TIE*, 2003.

[6] Park, Shim, & Joo, "Recovering nominal tracking performance in an asymptotic sense for uncertain linear systems," *SIAM*, 2018.

Mechanical positioning system

Computer Simulation

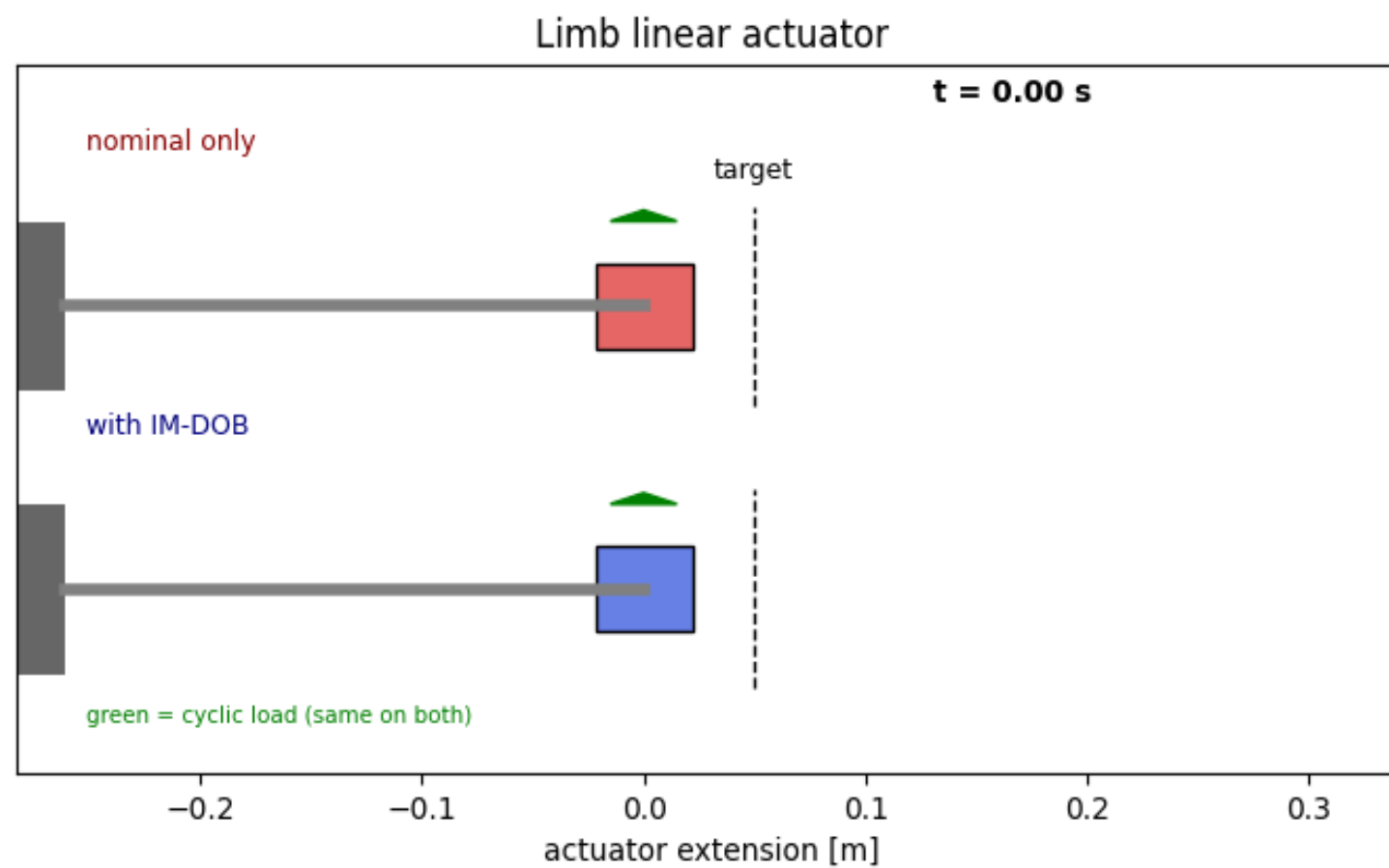
- Initial conditions: Set to all zero except for $w(0) \in [-1, 1]^4$.
- Time-scale parameter: $\tau = 0.005$.



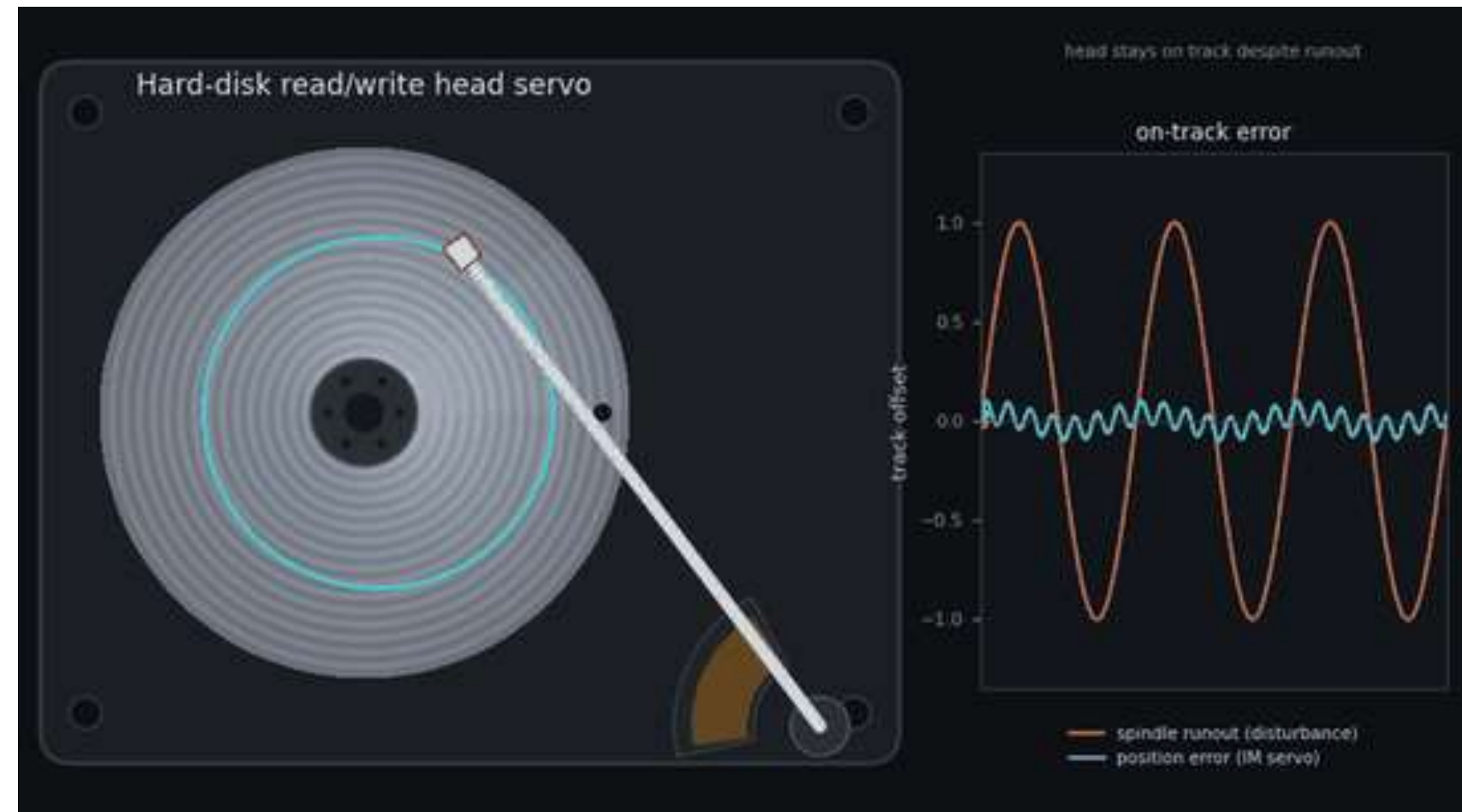
Note: $\hat{u}(t) = q_1(t) \rightarrow \Pi_1 w(t) = -Rw(t) = -d(t)$.

Real Systems that use disturbance observers

A linear actuator holds position against a **periodic load disturbance** (e.g. a gait-frequency cyclic force).



The read/write head must stay on a data track only tens of nanometres wide.



Conclusion

- We have designed an internal-model-based DOB for a class of linear systems, and shown the asymptotic stability of the closed-loop system.
- The matrix identity describing the internal model principle has played a crucial role.

$$\Pi S = (A_\mu - aC_\mu) \Pi, \quad -R = C_\mu \Pi.$$

- The existing internal-model-based design of DOBs handles the disturbance by incorporating it into the quasi-steady-states, whereas our method explicitly takes it into account in the coordinate changes using the developed matrix identity. By this, we have

$$\lim_{t \rightarrow \infty} \|q(t) - \Pi w(t)\| = 0$$

and the stability analysis has been simplified compared to the existing works.

Thank you!